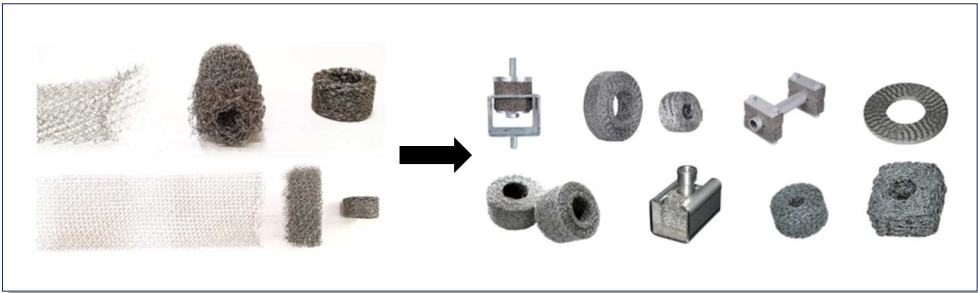




# Lifetime calculation for metal wire mesh vibration dampers subjected to cyclic loads

## Metal Cushion

- ❑ Metal cushions are derived from knitted wire (wire mesh) by pressing
- ❑ Applied in vibration engineering, when high damping, lifetime and robustness is needed [1]



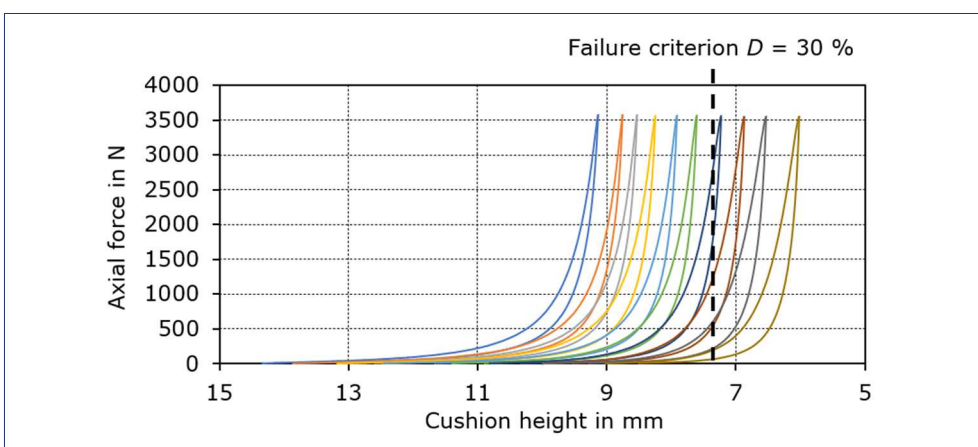
- ❑ Characteristic progressive force-deflection curve with hysteresis
- ❑ Full load capacity in one direction only (anisotropic material)
- ❑ Usually applied pre-tensioned in pairs as spring-damper system

## Damage / Failure Criterion

- ❑ Three damage mechanisms are identified:
  - Plastic deformation
  - Breakage of individual wires
  - Abrasion of the wires at the contact points



- ❑ Height of cushion is decreasing over the fatigue lifetime
- ❑ Determination of damage degree by recording characteristic curve in intervals throughout fatigue tests



- ❑ Damage degree described by increase of absolute deflection subjected to test load compared to initial condition
- ❑ Damage degree for a specific test force:  $D = \frac{h_0 - h_{p,t}}{h_0 - h_{p,0}} - 1$
- ❑ Exceeding the failure criterion of 30 % means no total failure of dampers

## Experimental

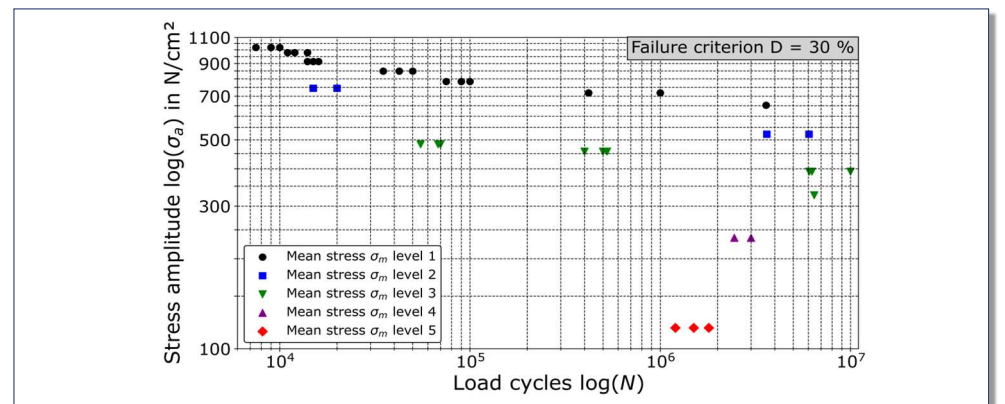
- ❑ Tests were carried out on pulse test bench with individual metal cushions
- ❑ Purely axial sinusoidal force with constant amplitude per test run



- ❑ Tests were terminated when failure criterion was reached or  $\geq 10^7$  cycles
- ❑ No influence of test frequency on fatigue was detected (5 to 100 Hz)

## Results

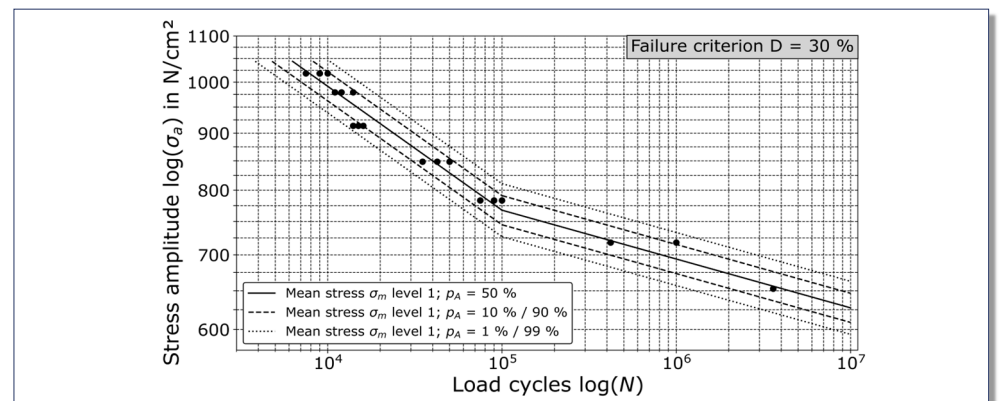
- ❑ Fatigue tests were performed at five levels of mean stress with different stress amplitudes
- ❑ Across all tests: The smaller the load and higher the lifetime, the more nonlinear is the progression of the damage degree



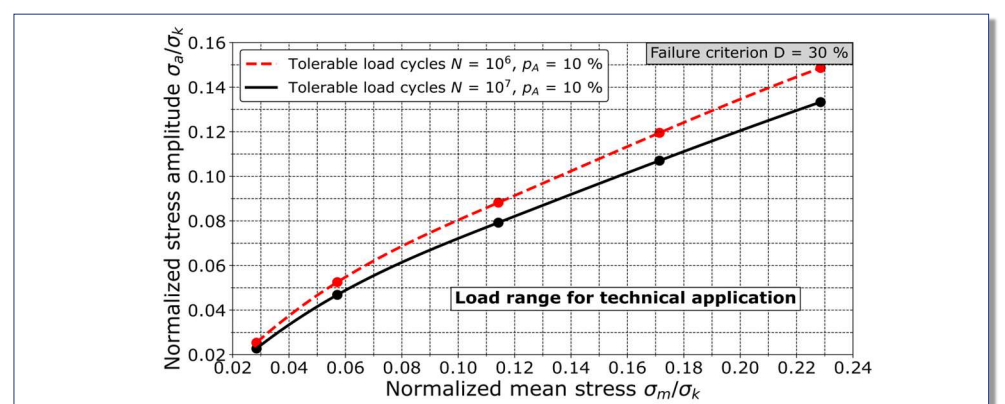
- ❑ Scatter of the test results increases with higher numbers of load cycles
- ❑ Fatigue lifetime is lowest, when the stress amplitude is close to the mean stress (low minimum stress)

## Validation / Utilization

- ❑ Results are statistically evaluated using a linear approach [2] [3]
- ❑ S-N curve with two slope ranges for each mean stress level (best fit)



- ❑ Failure probability  $p_A = 10\%$  acceptable for given failure criterion
- ❑ Fatigue safety factor  $j = 3$  suggested for metal cushion dimensioning



- ❑ Manufacturing surface pressure of a cushion is used as reference to normalize the stress and strength
- ❑ Mean stress sensitivity for different fatigue lifetimes can be used as limit curve for dimensioning of cushions

## Conclusions / Outlook

- ❑ Lifetime-oriented design of metal cushions in development is feasible
- ❑ Evaluation of S-N curves for metal cushions similar as for solid metals
- ❑ Linear damage accumulation with the obtained test data as a basis can be applied to calculate the damage degree of cushions due to load spectra
- ❑ Data of individual metal cushions can normalized be applied to systems

## Literature

- [1] A. Pérez et al., *Metal cushion dampers for railway applications: A review*, Construction and Building Materials, vol. 238, Mar. 2020
- [2] S. Einbock, *Betriebsfestigkeitsberechnung*, Norderstedt: Books on Demand, 2017
- [3] E. Haibach, *Betriebsfestigkeit Verfahren und Daten zur Bauteilberechnung*, Heidelberg: Springer Verlag, 2002